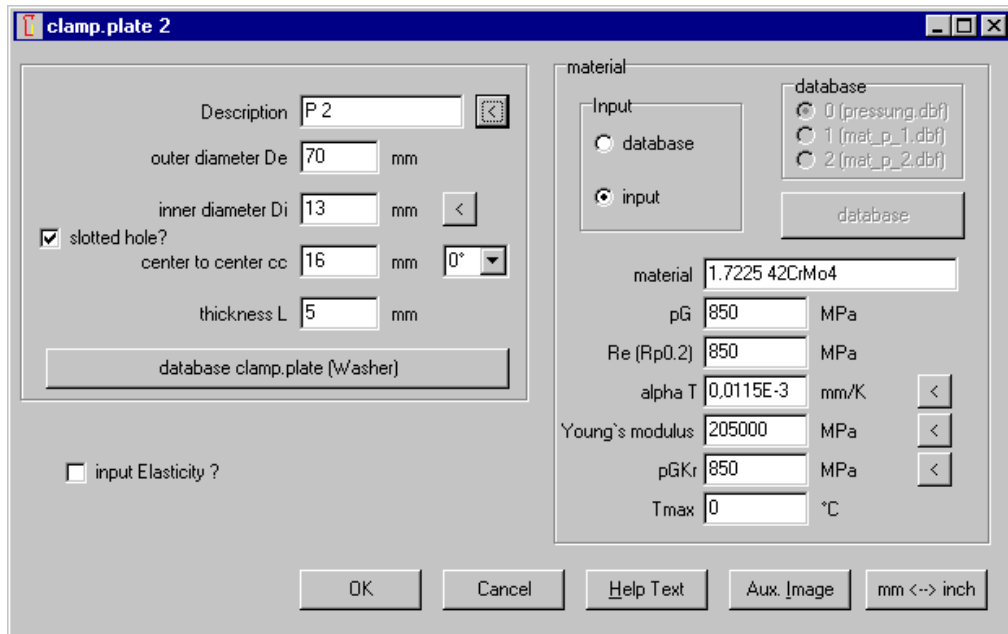
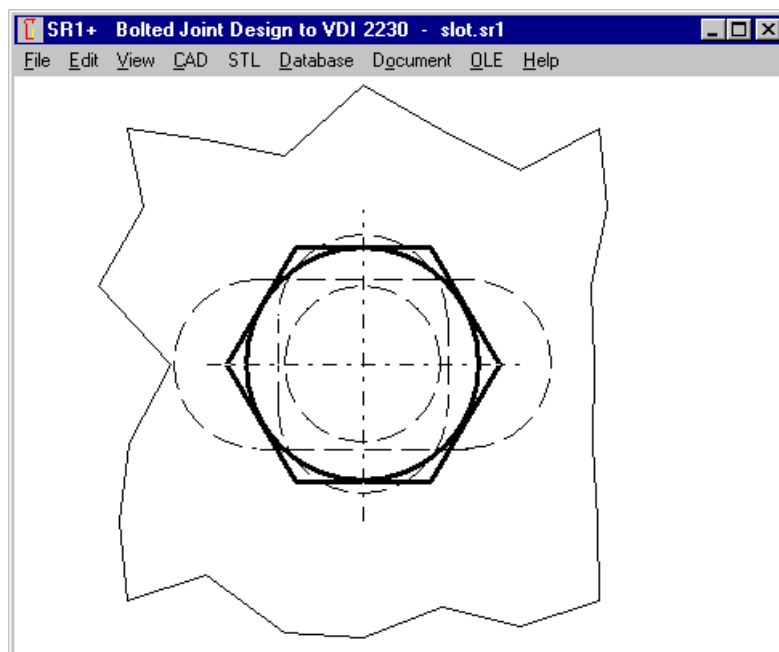


by Fritz Ruoss

**SR1+: Clamping plates with oblong hole**



An oblong hole instead of a round hole can now be defined for the clamping plates: slot width  $D_i$  and center-to-center distance  $cc$  ( $cc=0$  is round hole). Slot area is considered for calculation of maximum pressure (safety  $S_p$  and  $S_{pKr}$ ). Virtual diameter for round hole is used for deformation cone calculations. A position angle of  $0^\circ$  or  $90^\circ$  can be selected to calculate overlapping slot area of neighbored clamping plates with slots crossed or in same sense. Bolt position is always assumed in center of oblong hole.



## SR1+: Creep safety for FKRmin or FVmin

SR1+ calculation base

calculation base FM, MA  
 VDI 2230 : 1986  
 VDI 2230-1:2015

Elasticity  
 deformation sleeve (VDI 2230-1986)  
 deformation cone (VDI 2230-1:2015)

p max  
 deformation sleeve  
 deformation cone

dPzu (80) VDI 2230-1:2015

TTJ -> TBJ (phiD, dw nut)  
 D'A max = 10 dw

washer dwa=dw+1.6hs

creep at: FV min  
 thread length: FKR min  
 calc. min. thread length engag. for FSmax (=FMzul+FSA)  
 Tolerances for friction coefficients ?

tolerances d2, d3 for FM, MA ? max (d2=d2max, d3=d3max)

Multi-bolted joint (FA,FQ,FKR = f (MV) ?  
No Flange

calculation FA (Mb) flange  
 Dose, VDI2230-2 (34)  
 VDI2230-2 (43): Fmax=4\*Mb/(ns\*dt)

tightening angle incl. torsion bolt ?  
 TTJ: thread engagement mgeo and mtr reduced by bolt length tolerance

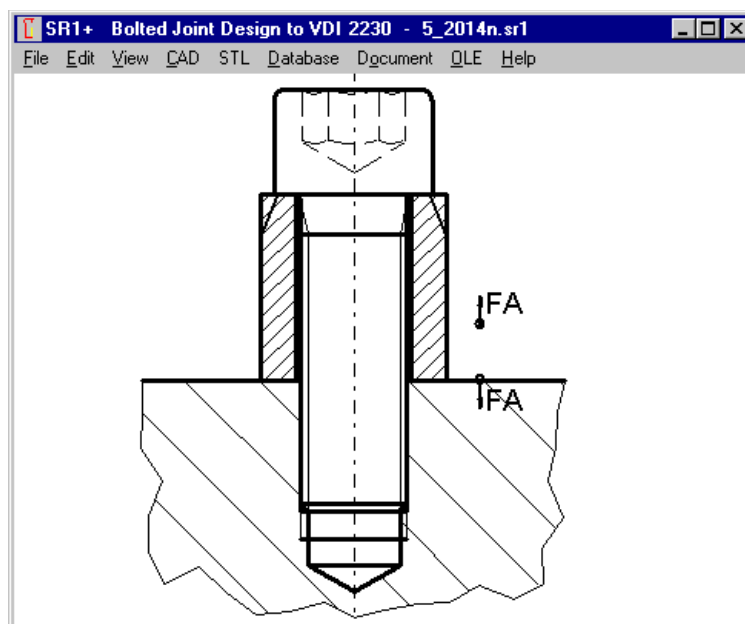
Units metric/imperial metric (mm, N, MPa, Nmm, °C)

OK Cancel Help Text Calc

Until now, creep safety was calculated based on FKRmin. For creep safety  $SpKr=1$ , creeping until residual clamp load FKRmin was allowed. For safer calculations, better allow creeping until minimum preload FVmin. Now you can configure if safety margin  $SpKr$  will be calculated with FKRmin or FVmin. Default setting is FVmin.

## SR1+: FA in drawing

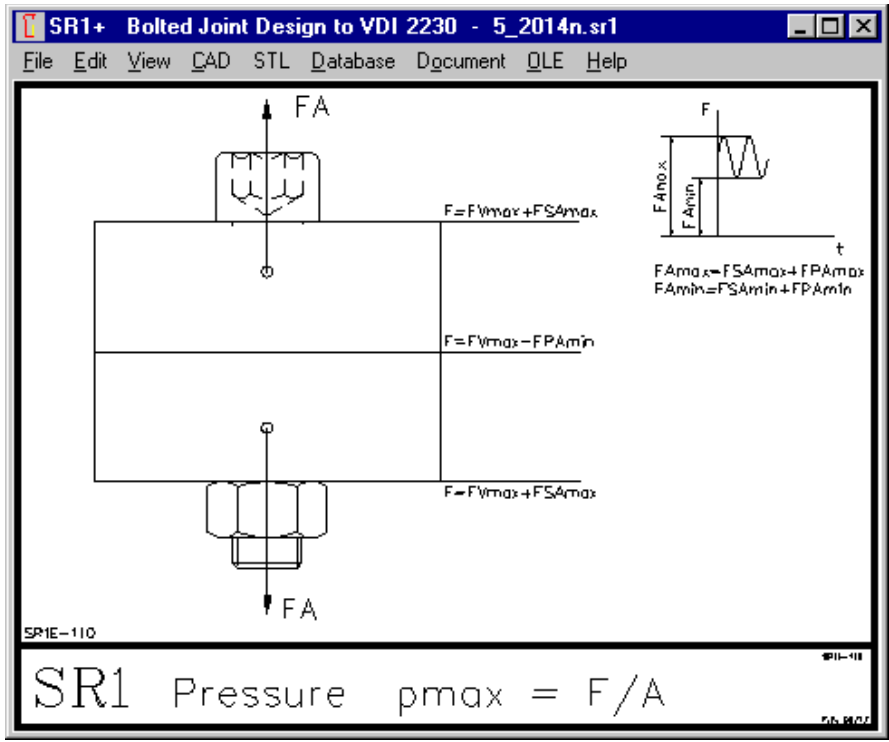
Axial load on clamping plates is plotted in the drawings now. With arrows as tensile force for  $F_{Amax} > 0$  and as compressive force for  $F_{Amin} < 0$ .



**SR1+: Maximum pressure in case of FA < 0 (pressure)**

The case that axial load FA may be compressive instead of tensile is not really defined in VDI 2230. According to VDI 2230-1:2015, maximum pressure in working state is  $p_{Bmax} = (FV_{max} + FS_{Amax} - \Delta FV_{th}) / A_{pmin}$  (formula 191). "For joints loaded in compression (negative FA),  $FS_{Amax} = 0$  is to be substituted."

SR1+ calculates more precise. Outside the selected clamping plates of load introduction, maximum pressure is calculated with  $FS_{Amax}$ , and inside FA introduction positions without  $FS_{Amax}$  (in assembly state). In case of compressive load, outside the load introduction positions, formula according to VDI 2230 is valid (with  $FS_A=0$ ). But inside the FA position, compressive load share  $FPA$  must be added.  $p_{Bmax} = (FV_{max} - FP_{Amin} - \Delta FV_{th}) / A_{pmin}$  (with  $FP_{Amin} < 0$ )



In case of  $FA_{max} > 0$  and  $FA_{min} < 0$  (alternating load), maximum pressure outside of FA introduction points is increased by  $FS_{Amax}$  (as described in VDI 2230), and inside FA introduction points it is increased by  $-FP_{Amin}$  (with  $FP_{Amin} < 0$ ).

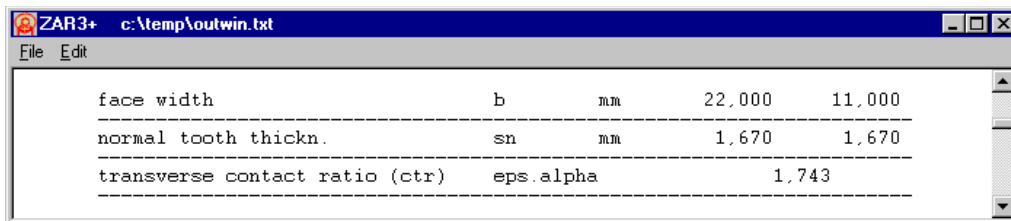
**SR1: Rp0.2 instead of Re**

Labels of some material database captions of yield point are not really correct: it should be "Rp0.2" instead of "Re". Titles of bolt and plate material databases have been changed from "RE" into "RP02". Data remain all the same.

IDENT	RP02	RM	BETA_B	E_MODUL	ALPHA_T	INFO1	INFO2	TAUB_RM	A5
12.9	1100	1220	0,577	210000	1,15E-5			0,6	8
11.9	1034	1172	0,577	210000	1,15E-5			0,61	8,5
10.9	940	1040	0,577	210000	1,15E-5			0,62	9
9.8	720	900	0,577	210000	1,15E-5			0,65	10
8.8 d<=16	640	800	0,577	210000	1,15E-5	d <= 16		0,65	12
8.8 d>16	660	830	0,577	210000	1,15E-5	d > 16		0,65	12

### ZAR3+: Contact ratio epsilon alpha

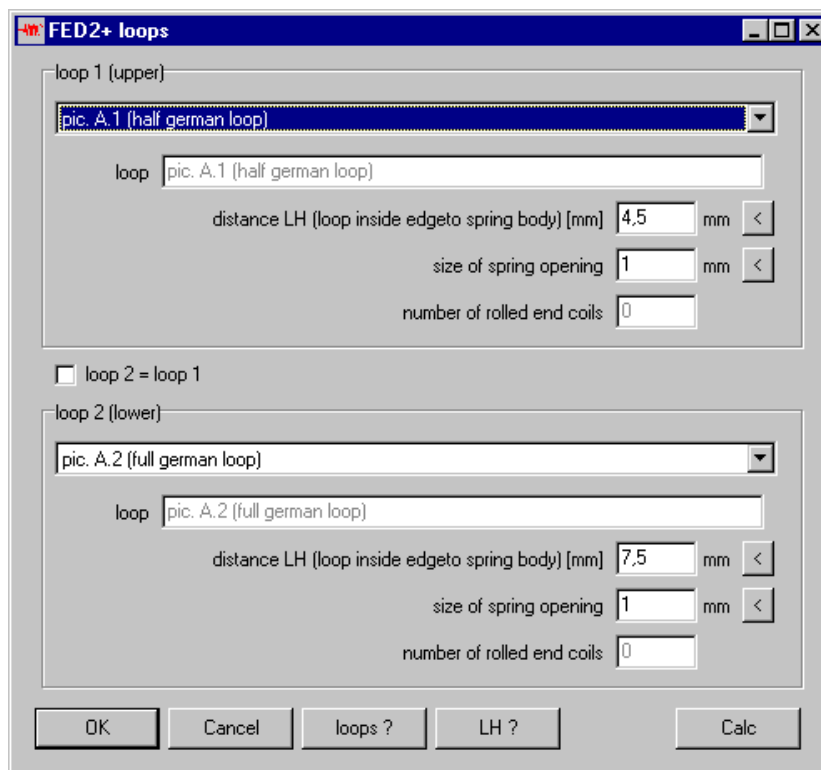
Transverse contact ratio in gear center and normal section has been added in printout for ZI worm flank type.



Parameter	Symbol	Unit	Value 1	Value 2
face width	b	mm	22,000	11,000
normal tooth thickn.	sn	mm	1,670	1,670
transverse contact ratio (ctr)	eps.alpha		1,743	

### FED2+: Suggest buttons at input of loops

New suggest buttons "<" for input of loop distance LH and size of spring opening have been added. FED2+ suggests input dependent of selected loop type and inner coil diameter Di.



loop 1 (upper)

pic. A.1 (half german loop)

loop pic. A.1 (half german loop)

distance LH (loop inside edgeto spring body) [mm] 4,5 mm <

size of spring opening 1 mm <

number of rolled end coils 0

loop 2 = loop 1

loop 2 (lower)

pic. A.2 (full german loop)

loop pic. A.2 (full german loop)

distance LH (loop inside edgeto spring body) [mm] 7,5 mm <

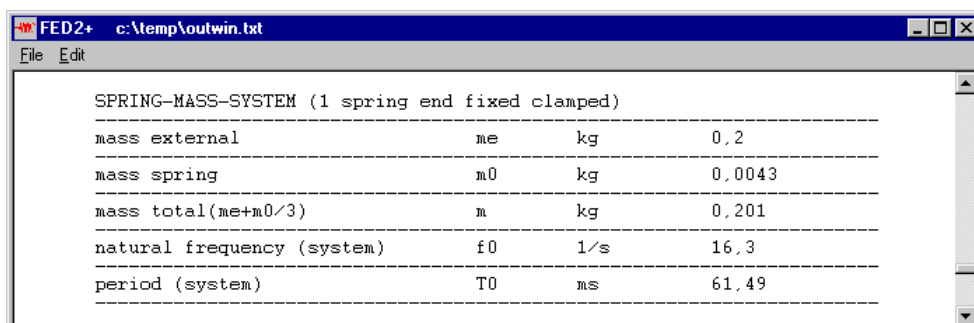
size of spring opening 1 mm <

number of rolled end coils 0

OK Cancel loops ? LH ? Calc

### FED1+, FED2+: Period time of spring-mass-system

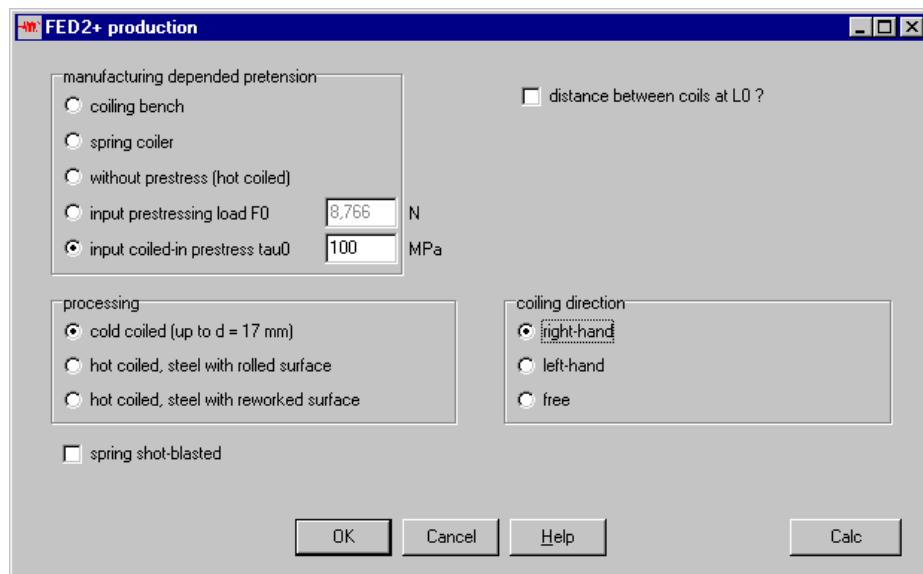
FED1+ and FED2+ calculates resonance frequency of spring-mass-system, if you input an external mass. Time required for one stroke (period  $T = 1/f$ ) in milliseconds has been added in the printout.



Parameter	Symbol	Unit	Value
mass external	me	kg	0,2
mass spring	m0	kg	0,0043
mass total (me+m0/3)	m	kg	0,201
natural frequency (system)	f0	1/s	16,3
period (system)	T0	ms	61,49

## FED2+: Input coiled-in prestress as load F0 or stress tau0

FED2+ calculates in-coiled prestress load F0 according to EN13906-2 (produced by coiling bench or spring coiler), or set to 0 (if distance between coils or hot-rolled), or input F0 directly. As additional option, you can enter prestress tau0 in MPa now.

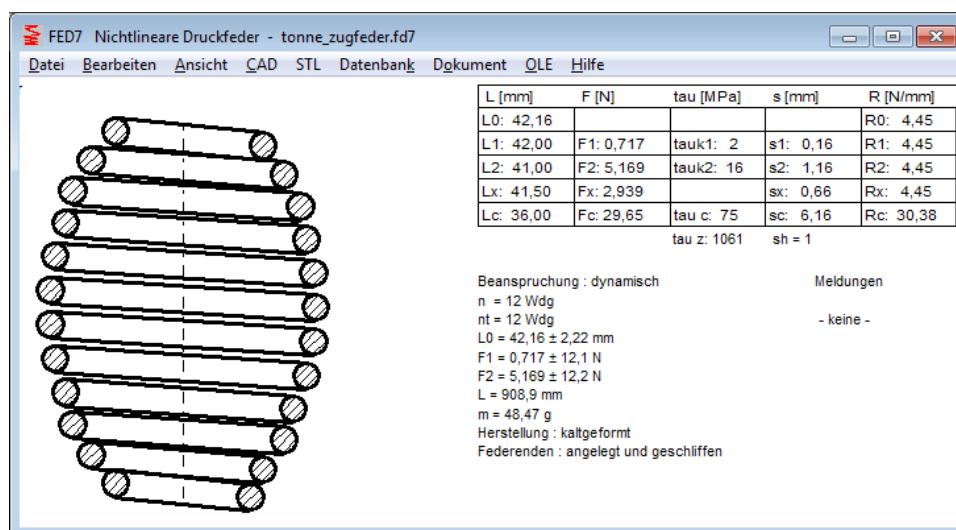


## Extension Springs in barrel shape or with conical end coils

Extension springs with large coil diameter are sometimes manufactured with decreasing coil diameter at the spring ends. The loop becomes small then. This favors reduced bending stress in the loop due to a small lever arm.

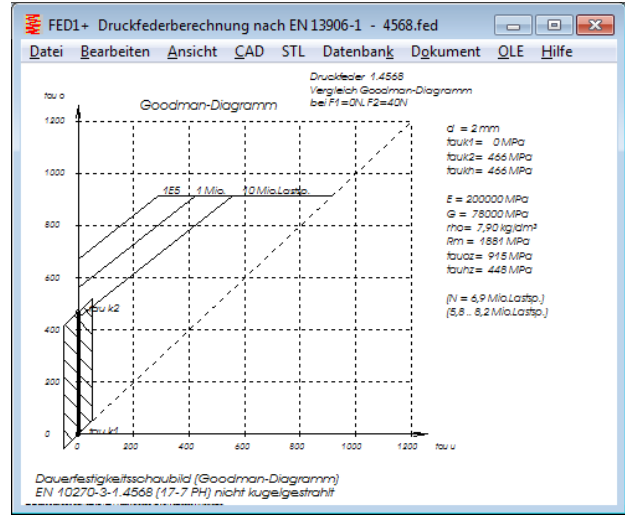
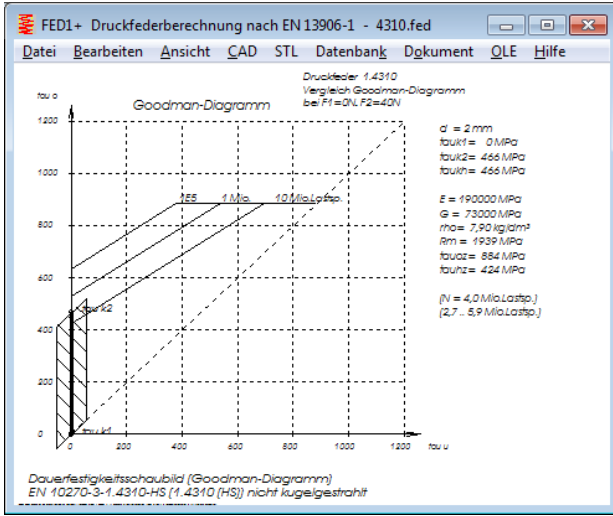
Load-extension diagram of extension springs with barrel shape or conical shape is not progressive, as wrongly asserted by Gutekunst (Hanser Konstruktion 5/2017). Load-extension diagram of these extension springs may even be slightly degressive, if coils of smaller coil diameter require higher load to overpower prestress tau0. FED7 can be used to calculate spring rate of extension springs of any shape: this is the spring rate R0. R0 of extension springs, however, is the spring rate at the end of the load-extension diagram, not at start as for compression springs.

If demanded, we can make a new software for this extension spring type „FED7 for extension springs“.

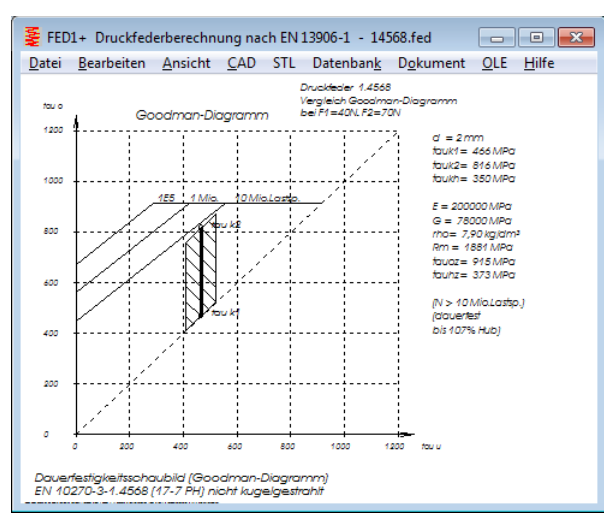
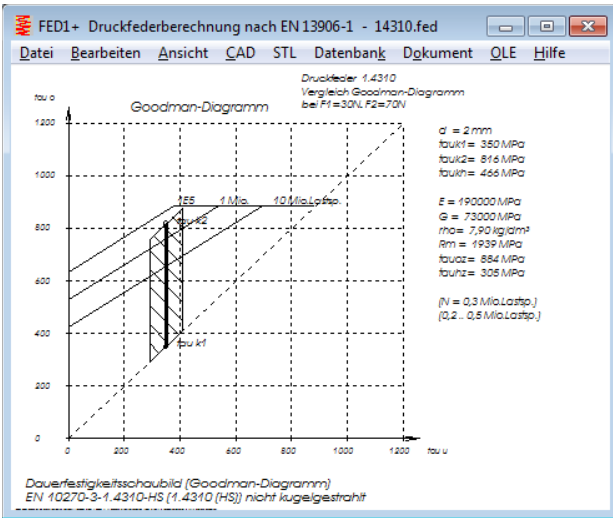


## Spring Calculation: Comparison of material 1.4310 (302) and 1.4568 (17-7 PH)

Customers asked why 1.568 should be better than 1.4310. From material properties, this cannot be seen. But shear modulus of 1.4568 (78000) is higher than G module of 1.4310 (73000). Static safety of 1.4568 or 1.4401 is not better than 1.4310. Not even fatigue strength safety is better, if pretension  $\tau_{k1}$  is small or 0.



But 1.4568 is better in fatigue strength if pretension  $\tau_{k1}$  is high. And 1.4568 is better if you compare fatigue strength for limited life (100,000 cycles).



Example calculation with  $F_1=30N$  and  $F_2=70N$  shows the difference by means of the Goodman diagram: spring made of 1.4568 is fatigue strength safe, but spring made of 1.4310 fails after 300,000 strokes.

## New EN 10270-1:2017

A new DIN EN 10270-1:2017 (Patented cold drawn unalloyed spring steel wire) was released. I found no essential changes. In table 3 with mechanical properties, minimum tensile strength for SM and DM with  $14 < d \leq 15mm$  was corrected. The value already was correct in our database file fedrmin.dbf (1110 MPa), nothing to change.

## TOL1 – Calculation of reject rate for pre-defined limits

Optional input of limit min/max has been added to the input of closing dimensions. TOL1 then calculates reject rate of the dimensions that lay outside the defined limits.

Standard printout will be shorter in this case: instead of reject rates for different dimension intervals, only the reject rate of the defined interval will be printed.

```

Closing dimensions for constant distribution
-----
Distanc  Nom dim.  Up.t.  L.tol  Max.clear  Min.clear  Comment
-----
1 2      1,000  0,810  -0,750    1,810    0,250  clearance
12 13     0,000  0,600  -0,200    0,600    -0,200  cover-case
12 7      64,000  0,560  -0,500    64,560    63,500  total length
5 6      3,000  0,660  -0,600    3,660    2,400  stroke
-----

Closing dimensions for Gaussian distribution
-----
Distanc  Mid dim.  Up.t.  L.tol  Max.clear  Min.clear  Comment
-----
1 2      1,030  0,358  -0,358    1,388    0,672  clearance
12 13     0,200  0,187  -0,187    0,387    0,013  cover-case
12 7      64,030  0,257  -0,257    64,287    63,773  total length
5 6      3,030  0,275  -0,275    3,305    2,755  stroke
-----

clearance
closing dim.: 1,030 +/- 0,358 at +/- 3,00 Sigma (0,27%)
limit value: 0 < x < 1,25
reject rate: 3,274 % (32738 ppm)
x < 0,000: 0 %
x > 1,250: 3,274 %
-----

cover-case
closing dim.: 0,200 +/- 0,187 at +/- 3,00 Sigma (0,27%)
limit value: 0 < x < 0,3
reject rate: 5,507 % (55071 ppm)
x < 0,000: 0,067 %
x > 0,300: 5,44 %
-----

total length
closing dim.: 64,030 +/- 0,257 at +/- 3,00 Sigma (0,27%)
limit value: 0 < x < 100
reject rate: 0 % (0 ppm)
x < 0,000: 0 %
x > 100,000: 0 %
-----

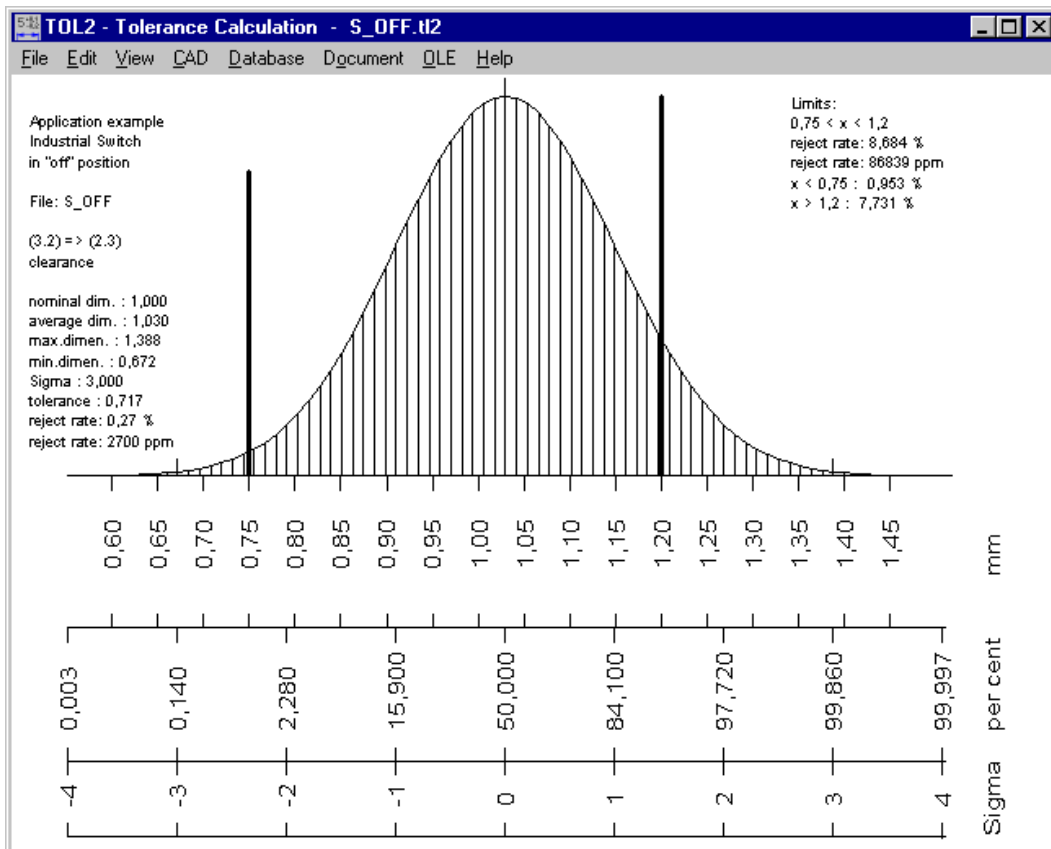
stroke
closing dim.: 3,030 +/- 0,275 at +/- 3,00 Sigma (0,27%)
limit value: 2,7 < x < 3,4
reject rate: 0,0191 % (191 ppm)
x < 2,700: 0,0163 %
x > 3,400: 0,00280 %
-----

```

In "Gaussian Graphic", self-defined dimension limit is plotted and reject rates relative to limits are printed.

## TOL2 – Calculation of reject rate for pre-defined limits

In the same way as in TOL1, you can now enter limits for each closing dimension, and TOL2 calculates reject rate. Standard printout lists reject rates for the predefined limits, and "Gaussian Graphic" plots predefined dimension limit and lists reject rates relative to limit.



### 30 years TOL1

In September 1987, the first HEXAGON software was released, TOL1 for tolerance calculation. Made by Ruoss engineering office, HEXAGON was founded in 1990. Followed by gear calculation software ZAR1 and spring calculation software FED1. Operating system was MS-DOS, and the programs were delivered on 360 kB floppy disc 5.25". And I was 28 years old. More 10 years, then my kids should take over HEXAGON software.



## HEXAGON PRICELIST 2017-09-01

PRODUCT	EUR
DI1 Version 1.2 O-Ring Seal Software	190,-
DXF-Manager Version 9.0	383,-
DXFPLOT V 3.2	123,-
FED1+ V29.6 Helical Compression Springs incl. spring database, animation, relax., 3D,..	695,-
FED2+ V20.4 Helical Extension Springs incl. spring database, animation, relaxation, ...	675,-
FED3+ V19.0 Helical Torsion Springs incl. prod.drawing, animation, 3D, rectang.wire, ...	480,-
FED4 Version 7.3 Disk Springs	430,-
FED5 Version 15.7 Conical Compression Springs	741,-
FED6 Version 16.3 Nonlinear Cylindrical Compression Springs	634,-
FED7 Version 13.2 Nonlinear Compression Springs	660,-
FED8 Version 6.9 Torsion Bar	317,-
FED9 Version 6.0 Spiral Spring	394,-
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FED12 Version 2.4 Elastomere Compression Spring	220,-
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FED15 Version 1.4 Leaf Spring (simple)	180,-
FED16 Version 1.1 Constant Force Spring	225,-
FED17 Version 1.0 Magazine Spring	725,-
GEO1+ V6.1 Cross Section Calculation incl. profile database	294,-
GEO2 V2.6 Rotation Bodies	194,-
GEO3 V3.3 Hertzian Pressure	205,-
GEO4 V4.2 Cam Software	265,-
GEO5 V1.0 Geneva Drive Mechanism Software	218,-
GR1 V2.0 Gear construction kit software	185,-
HPGL-Manager Version 9.0	383,-
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LG2 V2.2 Hydrodynamic Plain Journal Bearings	460,-
SR1 V22.5 Bolted Joint Design	640,-
SR1+ V22.5 Bolted Joint Design incl. Flange calculation	750,-
TOL1 V12.0 Tolerance Analysis	506,-
TOL2 Version 4.0 Tolerance Analysis	495,-
TOLPASS V4.1 Library for ISO tolerances	107,-
TR1 V4.0 Girder Calculation	757,-
WL1+ V20.1 Shaft Calculation incl. Roll-contact Bearings	945,-
WN1 Version 11.6 Cylindrical and Conical Press Fits	485,-
WN2 V10.0 Involute Splines to DIN 5480	250,-
WN2+ V10.0 Involute Splines to DIN 5480 and non-standard involute splines	380,-
WN3 V 5.4 Parallel Key Joints to DIN 6885, ANSI B17.1, DIN 6892	245,-
WN4 V 4.6 Involute Splines to ANSI B 92.1	276,-
WN5 V 4.6 Involute Splines to ISO 4156 and ANSI B 92.2 M	255,-
WN6 V 3.0 Polygon Profiles P3G to DIN 32711	180,-
WN7 V 3.0 Polygon Profiles P4C to DIN 32712	175,-
WN8 V 2.2 Serration to DIN 5481	195,-
WN9 V 2.2 Spline Shafts to DIN ISO 14	170,-
WN10 V 4.1 Involute Splines to DIN 5482	260,-
WN11 V 1.3 Woodruff Key Joints	240,-
WNXE V 2.0 Involute Splines - dimensions, graphic, measure	375,-
WNXK V 2.0 Serration Splines - dimensions, graphic, measure	230,-
WST1 V 10.02 Material Database	235,-
ZAR1+ V 26.0 Spur and Helical Gears	1115,-
ZAR2 V7.9 Spiral Bevel Gears to Klingelberg	792,-
ZAR3+ V9.0 Cylindrical Worm Gears	620,-
ZAR4 V5.2 Non-circular Spur Gears	1610,-
ZAR5 V11.5 Planetary Gearings	1355,-
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ZAR7 V1.4 Plus Planetary Gears	1380,-
ZAR8 V1.4 Ravigneaux Planetary Gears	1950,-

ZARXP V2.1 Involute Profiles - dimensions, graphic, measure	275,-
ZAR1W V1.7 Gear Wheel Dimensions, tolerances, measure	450,-
ZM1.V2.5 Chain Gear Design	326,-

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<b>HEXAGON Mechanical Engineering Package</b> (TOL1, ZAR1+, ZAR2, ZAR3+, ZAR5, ZAR6, WL1+, WN1, WN2+, WN3, WST1, SR1+, FED1+, FED2+, FED3+, FED4, ZARXP, TOLPASS, LG1, DXFPLOT, GEO1+, TOL2, GEO2, GEO3, ZM1, WN6, WN7, LG2, FED12, FED13, WN8, WN9, WN11, DI1, FED15, WNXE, GR1)	8,500.-
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